

# Adaptive Optimal Control Based on Critic-Actor Architecture for Hydraulic Support Cylinder System with Asymmetric Output Error Constraints

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**Abstract**—This paper takes the hydraulic support cylinder system (HSCS) as the research object. Based on its working principle and physical characteristics, a nonlinear mathematical model describing the electro-hydraulic control power system of the hydraulic support is constructed. Firstly, by combining reinforcement learning technology and asymmetric output constraint theory, an adaptive asymmetric output constraint optimal control strategy is proposed, aiming to enhance the robustness of the system and meet industrial constraint requirements. Secondly, by adopting a simplified optimal backstepping design method, an adaptive optimal controller is designed to ensure that the performance of each subsystem reaches the optimum and the output error of the system is always limited within the preset asymmetric constraint range. Finally, the simulation verification shows that the proposed method has good effectiveness and engineering feasibility.

**Index Terms**—Hydraulic support cylinder, Adaptive optimal control, Reinforcement learning, Asymmetric output constraints

## I. INTRODUCTION

AS the tide of industrial intelligence sweeps across the landscape, the intelligent control system governing the “trinity” of fully mechanized mining operations has emerged as a cornerstone technology for safeguarding safety and amplifying efficiency. Among its critical elements, the electro-hydraulic control cylinder of the hydraulic support stands out, yet it grapples with formidable challenges owing to its pronounced nonlinear hysteresis and parameters that vary with time. These hurdles include the intricacies of dynamic modeling and the shortfall in servo tracking precision [1, 2]. Fortunately, the evolution of adaptive nonlinear control theory heralds an ingenious resolution, paving the way for the intelligent advancement of such sophisticated industrial systems [3, 4].

Optimal control for nonlinear systems is one of the core aspects of modern control theory, aiming to optimize the performance indicators of control systems [5]. It integrates the fundamental conditions and methods distilled from practical problems, with the research object being controlled dynamic systems or motion processes. It seeks the best control scheme among the allowable ones to ensure the system achieves the optimal performance when transitioning from the initial state to the target state [6]. With the rapid development of digital technology and electronic

computers, optimal control has been widely applied in production, military, and economic activities, playing a significant role in the national economy and national defense. The optimal problem is theoretically equivalent to solving the Hamilton-Jacobi-Bellman (HJB) equation [7], but due to its strong nonlinearity and dynamic uncertainty, it is difficult to solve directly through analysis. To overcome this challenge, reinforcement learning (RL) and adaptive dynamic programming (ADP) have been proven to be effective solutions. RL and ADP were initially proposed by Werbos for discrete systems [8] and later extended to continuous systems [9, 10], but they are only applicable to affine nonlinear systems. For the control problem of nonlinear mismatched systems, in [11] proposed an optimal control method based on the backstepping framework, ensuring the optimization of each subsystem. To simplify complexity and alleviate the continuous excitation condition, in [12–14] further simplified the optimal backstepping control strategy.

Constraints are a frequent occurrence in practical control systems. Considering that control systems often have to strike a balance between performance requirements and physical limitations, the constraint problem holds significant importance in practical control systems [15, 16]. To date, the barrier Lyapunov function (BLF) has become the mainstream method for dealing with constraints due to its ability to constrain state variables within a predefined compact set [17–21]. In [17], the BLF method was used to control a multi-input multi-output (MIMO) nonlinear system to achieve practical stability under output constraint conditions. In [18], a neural network adaptive fault-tolerant controller based on integral type BLF was adopted, and it was proposed that the full state constraint of the uncertain nonlinear system could be satisfied even if the initial value violated the predefined compact set limit. However, it should be noted that the BLF technique in [17, 18] is only used to handle symmetric constraints. Recently, in [19–21], further research on the tracking control of nonlinear systems under time-varying asymmetric constraint conditions has been conducted using different types of BLF. That is, the constraints are allowed to be asymmetric and time-varying, and the boundary conditions have been greatly relaxed in these works, which is more in line with the application of actual systems.

Based on the above research, this paper takes the working principle of the hydraulic support electro-hydraulic control cylinder as the control object, and combines the reinforcement learning algorithm and adaptive control theory, aiming to optimize the overall stability and robustness of the system operation. Through the modeling analysis, controller

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design and simulation verification of the working process of the hydraulic support electro-hydraulic control cylinder system, an optimal control strategy combining reinforcement learning and asymmetric output constraints is proposed. The specific contributions are summarized as follows: 1) Mathematical analysis and modeling of the working principle and physical process of Hydraulic support cylinder; 2) Keeping all signals in the Hydraulic support cylinder system bounded while achieving performance optimization of each subsystem; 3) The system output error strictly conforms to the pre-designed form and avoids any violation of constraints throughout the process.

## II. MODEL DESCRIPTION AND PRELIMINARIES

Assuming that the electro-hydraulic control cylinder system of the hydraulic support operates in the direction shown in Figure 1 during operation, according to the force balance equation and the flow balance equation, its dynamic equation can be expressed as:

$$M\ddot{X}_p = p_1 A_1 - p_2 A_2 - B\dot{X}_p + F_T \quad (1)$$

where  $p_1 = (p_s + p_r)/2$ ,  $p_2 = (p_s - p_r)/2$ .  $m$  is the load mass,  $X_p$  is the displacement of the hydraulic support cylinder,  $B$  is the damping coefficient, and  $F_T$  is the external force acting on the hydraulic support cylinder.  $A_1$  and  $A_2$  are the effective areas of the non-symmetric cylinder's rodless chamber and rod chamber respectively.  $p_1$  and  $p_2$  are the pressures at the oil cylinder's inlet and outlet respectively.  $p_s$  and  $p_r$  are the supply and return oil pressures respectively.

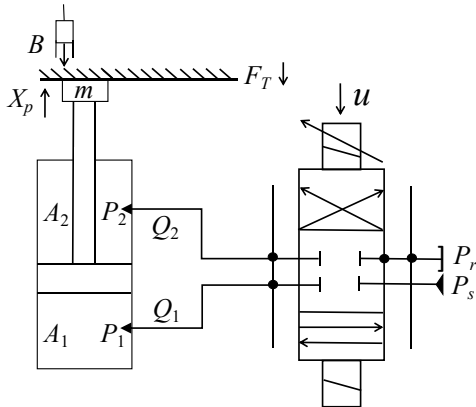


Fig. 1: Model diagram of hydraulic support cylinder.

Subsequently, by introducing a state space transformation  $\xi_1 = mX_p$ ,  $\xi_2 = m\dot{X}_p$ , and  $\xi_3 = p_1 A_1 - p_2 A_2$ , (1) can be transformed into a nonlinear system of the following form:

$$\begin{cases} \dot{\xi}_1 = \xi_2 \\ \dot{\xi}_2 = \xi_3 - \frac{B}{m}\xi_2 - F_T - mg \\ \dot{\xi}_3 = \gamma_1 u - \gamma_2 \xi_2 - \gamma_3(p_1 - p_2) \\ y = \xi_1 \end{cases} \quad (2)$$

where  $\xi = [\xi_1, \xi_2, \xi_3]^T \in \mathbb{R}^3$  denotes the state variables.  $u$  and  $y$  are the control input and output, respectively. The  $u$  is a voltage signal ranging from 0 – 10 V, which satisfies the linear relationship  $x_v = k_v u$ . In this paper,  $x_v$  denotes the

displacement of the spool in the proportional valve, and  $k_v$  represents a positive constant.

Then, for  $\forall t \geq t_0$ , the system output is constrained by a asymmetric time-varying boundary condition, defined as follows:

$$-\underline{\varrho}_{a1}(t) \leq \xi_1 \leq \bar{\varrho}_{a1}(t) \quad (3)$$

where  $\underline{\varrho}_{a1}(t)$  and  $\bar{\varrho}_{a1}(t)$  represent time-varying boundary functions, with the constraint being asymmetric when  $\underline{\varrho}_{a1}(t) \neq \bar{\varrho}_{a1}(t)$ .

Additionally, the bounded parameters  $\gamma_1$ ,  $\gamma_2$ , and  $\gamma_3$  are defined as follows:

$$\begin{cases} \gamma_1 = \left( \frac{A_1 R_1}{V_1 + \frac{A_1}{m}\xi_1} + \frac{A_2 R_2}{V_2 - \frac{A_2}{m}\xi_1} \right) \beta_e k_q k_v \\ \gamma_2 = \left( \frac{A_1^2}{mV_1 + A_1 \xi_1} + \frac{A_2^2}{mV_2 - A_2 \xi_1} \right) \beta_e \\ \gamma_3 = \left( \frac{A_1}{V_1 + \frac{A_1}{m}\xi_1} + \frac{A_2}{V_2 - \frac{A_2}{m}\xi_1} \right) \beta_e C_t \end{cases} \quad (4)$$

where  $R_1 = \sqrt{p_s + \text{sign}(x_v)(p_s - 2p_1)}$  and  $R_2 = \sqrt{p_s + \text{sign}(x_v)(2p_2 - p_s)}$ .  $\beta_e$  represents the effective volume elastic modulus of the hydraulic system,  $C_t$  is the leakage coefficient within the hydraulic cylinder,  $K_q$  is the flow gain of the proportional valve, and  $V_1$  and  $V_2$  are the initial volumes of the two chambers of the hydraulic cylinder.

**Lemma 1** [22] Let  $f(x)$  be a continuous function defined on a compact set  $\Omega_x$ . Then for  $\forall \varepsilon > 0$ , there exist the NN  $\zeta^T \Psi(x)$  such that

$$\sup_{x \in \Omega_x} |f(x) - \zeta^T \Psi(x)| \leq \varepsilon \quad (5)$$

where  $\zeta = [\zeta_1, \zeta_2, \dots, \zeta_m]^T \in \mathbb{R}^m$  is the weight vector and  $\Psi(x) = [\psi_1(x), \psi_2(x), \dots, \psi_m(x)]^T$  is the NN basis function with  $m > 1$  is the number of NN rules.  $\psi_i(x) = \exp[-\|x - \xi_i\|^2 / \vartheta_i^2]$ ,  $i = 1, 2, \dots, m$  is the Gaussian function, where  $\vartheta_i$  and  $\xi_i = [\xi_{i1}, \xi_{i2}, \dots, \xi_{im}]^T$  represent the width and center, respectively. The optimal parameter vector  $\zeta^*$  of NN is defined as

$$\zeta^* = \arg \min_{\zeta \in \mathbb{R}^m} \left\{ \sup_{x \in \Omega_x} |f(x) - \zeta^T \Psi(x)| \right\} \quad (6)$$

Therefore, the continuous function  $f(x)$  can be expressed as

$$f(x) = \zeta^{*T} \Psi(x) + \varepsilon(x) \quad (7)$$

where  $\varepsilon(x)$  is the NN approximation error, which can be bounded by  $|\varepsilon(x)| \leq \bar{\varepsilon}$ , where  $\bar{\varepsilon}$  is a positive constant. It should be pointed out that since  $\zeta^*$  is an analytical quantity, it needs to be estimated later for practical use.

**Assumption 1.** There exist time-varying functions  $\underline{\varrho}_{b1}(t)$  and  $\bar{\varrho}_{b1}(t)$  such that the reference signal  $y_r$  satisfies  $-\underline{\varrho}_{a1}(t) < \underline{\varrho}_{b1}(t) \leq y_r \leq \bar{\varrho}_{b1}(t) < \bar{\varrho}_{a1}(t)$ . Additionally, the derivatives of  $-\underline{\varrho}_{a1}(t)$ ,  $\underline{\varrho}_{b1}(t)$ ,  $y_r$ ,  $\bar{\varrho}_{b1}(t)$ , and  $\bar{\varrho}_{a1}(t)$  are known and bounded.

## III. MAIN RESULT

In this section, we will combine the reinforcement learning algorithm and the asymmetric output constraint control method to design an optimal backstepping control strategy under the critic-actor architecture, thereby constructing an optimal controller.

### A. Optimized backstepping design

First, consider the following tracking error coordinate transformation:

$$\begin{aligned} z_1 &= \xi_1 - y_r \\ z_2 &= m\xi_2 - \hat{\alpha}_1^* \\ z_3 &= \xi_3 - \hat{\alpha}_2^* \end{aligned} \quad (8)$$

where  $y_r$  is selected as the reference signal and set to  $0.2\sin(t)$ .  $\alpha_{i-1}(i = 2, 3)$  and  $\hat{\alpha}_{i-1}^*$  represent the virtual control and actual optimal virtual control correspondingly.

**Step 1:** In order to impose the asymmetric time-varying constraints on the output, an asymmetric Lyapunov function as given in [19] is introduced:

$$\check{V} = \frac{z_1^2}{(\bar{\phi}_1(t) - z_1)(\underline{\phi}_1(t) + z_1)} \quad (9)$$

where  $\underline{\phi}_1(t)$  and  $\bar{\phi}_1(t)$  are positive barrier functions. Define set  $\phi_{z_1} = \{-\underline{\phi}_1(t) < z_1 < \bar{\phi}_1(t)\}$ , ensuring that  $\check{V}$  is well-defined within this compact set. Simultaneously, the boundary functions designated as  $\underline{\phi}_1(t) = \underline{\phi}_{a_1}(t) - \underline{\phi}_{b_1}(t)$  and  $\bar{\phi}_1(t) = \bar{\phi}_{a_1}(t) - \bar{\phi}_{b_1}(t)$  are chosen to satisfy condition (3).

Then, the derivative of  $\check{V}$  is as follows:

$$\dot{\check{V}} = Pz_1(\dot{z}_1 + Q) \quad (10)$$

where

$$P = \frac{2\phi_1\bar{\phi}_1 - \phi_1z_1 + \bar{\phi}_1z_1}{[(\bar{\phi}_1 - z_1)(\underline{\phi}_1 + z_1)]^2} \quad (11)$$

$$Q = \frac{-\dot{\phi}_1\bar{\phi}_1 - \phi_1\dot{\bar{\phi}}_1 + z_1(\dot{\phi}_1 - \dot{\bar{\phi}}_1)}{2\phi_1\bar{\phi}_1 - \phi_1z_1 + \bar{\phi}_1z_1} \quad (12)$$

From (4) and (8), the derivative of  $z_1$  can be calculated

$$\dot{z}_1 = m\xi_2 - \dot{y}_r \quad (13)$$

The optimal performance index function is chosen as

$$J_1(z_1) = \int_t^\infty h_1(z_1(v), \alpha_1(z_1(v))) dv \quad (14)$$

where  $h_1(z_1, \alpha_1) = z_1^2 + \alpha_1^2$  is the cost function, and let the optimal virtual control  $\alpha_1^*$  replace  $\alpha_1$  in (13), the optimal performance index function can be obtained

$$\begin{aligned} J_1^*(z_1) &= \int_t^\infty h_1(z_1(v), \alpha_1^*(z_1(v))) dv \\ &= \min_{\alpha_1 \in \Omega_{z_1}} \left\{ \int_t^\infty h_1(z_1(v), \alpha_1(z_1(v))) dv \right\} \end{aligned} \quad (15)$$

Replace  $\xi_2$  in (12) with the optimal virtual control  $\alpha_1^*$ , and subsequently define the HJB equation associated with (12) and (14) as

$$H_1(z_1, \alpha_1^*, \frac{dJ_1^*}{dz_1}) = z_1^2 + \alpha_1^{*2} + \frac{dJ_1^*}{dz_1}(\alpha_1^* - \dot{y}_r) = 0 \quad (16)$$

The optimal virtual control  $\alpha_1^*$  can be computed by solving  $\partial H_1 / \partial \alpha_1^* = 0$  as

$$\alpha_1^* = -\frac{1}{2} \frac{dJ_1^*(z_1)}{dz_1} \quad (17)$$

Then,  $\frac{dJ_1^*(z_1)}{dz_1}$  is decomposed into

$$\frac{dJ_1^*(z_1)}{dz_1} = \left(\frac{2\chi_1}{P} + \frac{7P}{2}\right)z_1 + J_1^o(\xi_1, z_1) \quad (18)$$

where  $\chi_1 > 0$  is design parameter.  $J_1^o(\xi_1, z_1) = -(\frac{2\chi_1}{P} + \frac{7P}{2})z_1 + \frac{dJ_1^*(z_1)}{dz_1} \in \mathbb{R}$  is a continuous function, and substituting (18) into (17) has

$$\alpha_1^* = -\left(\frac{\chi_1}{P} + \frac{7P}{4}\right)z_1 - \frac{1}{2}J_1^o(\xi_1, z_1) \quad (19)$$

Since  $J_1^o(\xi_1, z_1)$  is continuous unknown function, it can be approximated by NN as follows:

$$J_1^o(\xi_1, z_1) = \zeta_{J_1}^{*T} \Psi_{J_1}(\xi_1, z_1) + \varepsilon_{J_1}(\xi_1, z_1) \quad (20)$$

where  $\zeta_{J_1}^*$  represents the ideal weight vector,  $\Psi_{J_1}(\xi_1, z_1)$  is the basis function vector, and  $\varepsilon_{J_1}(\xi_1, z_1)$  represents the approximation error bounded by  $|\varepsilon_{J_1}(\xi_1, z_1)| \leq \bar{\varepsilon}_{J_1}$  as arbitrarily small. Then, (18) and (19) can be reorganized as

$$\frac{dJ_1^*(z_1)}{dz_1} = \left(\frac{2\chi_1}{P} + \frac{7P}{2}\right)z_1 + \zeta_{J_1}^{*T} \Psi_{J_1} + \varepsilon_{J_1} \quad (21)$$

$$\alpha_1^* = -\left(\frac{\chi_1}{P} + \frac{7P}{4}\right)z_1 - \frac{1}{2}\zeta_{J_1}^{*T} \Psi_{J_1} - \frac{1}{2}\varepsilon_{J_1} \quad (22)$$

Since  $\zeta_{J_1}^*$  is unknown constant vector, the optimal virtual control (22) is not available for the controlled system. To derive the effective optimized virtual control, the following RL algorithm with critic and actor is performed.

$$\frac{d\hat{J}_1^*(z_1)}{dz_1} = \left(\frac{2\chi_1}{P} + \frac{7P}{2}\right)z_1 + \hat{\zeta}_{c1}^T \Psi_{J_1} \quad (23)$$

$$\hat{\alpha}_1^* = -\left(\frac{\chi_1}{P} + \frac{7P}{4}\right)z_1 - \frac{1}{2}\hat{\zeta}_{a1}^T \Psi_{J_1} \quad (24)$$

where  $\frac{d\hat{J}_1^*(z_1)}{dz_1}$  and  $\hat{\alpha}_1^*$  are the estimates of  $\frac{dJ_1^*(z_1)}{dz_1}$  and  $\alpha_1^*$ , respectively.  $\hat{\zeta}_{c1}^T \Psi_{J_1}$  and  $\hat{\zeta}_{a1}^T \Psi_{J_1}$  are the NN weight vectors of critic and actor, respectively.

Following this, the weight vectors of the neural networks for both the critic and actor are trained according to the respective adaptive laws outlined below.

$$\dot{\hat{\zeta}}_{c1} = -\kappa_{c1} \Psi_{J_1} \Psi_{J_1}^T \hat{\zeta}_{c1} \quad (25)$$

$$\dot{\hat{\zeta}}_{a1} = -\Psi_{J_1} \Psi_{J_1}^T (\kappa_{a1}(\hat{\zeta}_{a1} - \hat{\zeta}_{c1}) + \kappa_{c1} \hat{\zeta}_{c1}) \quad (26)$$

where  $\kappa_{c1} > 0$  and  $\kappa_{a1} > 0$  represent critic and actor design parameters, while  $\kappa_{c1}$  and  $\kappa_{a1}$  satisfy  $\kappa_{a1} > \frac{1}{2}$ ,  $\kappa_{a1} > \frac{\kappa_{c1}}{2}$ .

Using (24), (13) can be rewritten as

$$\dot{z}_1 = z_2 - \left(\frac{\chi_1}{P} + \frac{7P}{4}\right)z_1 - \frac{1}{2}\hat{\zeta}_{a1}^T \Psi_{J_1} - \dot{y}_r \quad (27)$$

For the first backstepping step, the Lyapunov function  $V_1$  is designed as follows:

$$V_1 = \check{V} + \frac{1}{2}\tilde{\zeta}_{c1}^T \tilde{\zeta}_{c1} + \frac{1}{2}\tilde{\zeta}_{a1}^T \tilde{\zeta}_{a1} \quad (28)$$

where  $\tilde{\zeta}_{c1} = \zeta_{J_1}^* - \hat{\zeta}_{c1}$  and  $\tilde{\zeta}_{a1} = \zeta_{J_1}^* - \hat{\zeta}_{a1}$  are the estimation errors of the critic and the actor, respectively.

Then, the derivative of  $V_1$  is

$$\begin{aligned} \dot{V}_1 &= Pz_1 \left( z_2 - \left(\frac{\chi_1}{P} + \frac{7P}{4}\right)z_1 - \frac{1}{2}\hat{\zeta}_{a1}^T \Psi_{J_1} - \dot{y}_r + Q \right) \\ &\quad + \kappa_{c1} \tilde{\zeta}_{c1}^T \Psi_{J_1} \Psi_{J_1}^T \hat{\zeta}_{c1} + \tilde{\zeta}_{a1}^T \Psi_{J_1} \Psi_{J_1}^T (\kappa_{a1}(\hat{\zeta}_{a1} - \hat{\zeta}_{c1}) + \kappa_{c1} \hat{\zeta}_{c1}) \end{aligned} \quad (29)$$

The Young's inequality yields the following results

$$\begin{aligned} Pz_1z_2 &\leq \frac{1}{2}P^2z_1^2 + \frac{1}{2}z_2^2 \\ PQz_1 &\leq \frac{1}{2}P^2z_1^2 + \frac{1}{2}Q^2 \\ -Pz_1\dot{y}_r &\leq \frac{1}{2}P^2z_1^2 + \frac{1}{2}\dot{y}_r^2 \\ -\frac{1}{2}Pz_1\hat{\zeta}_{a1}^T\Psi_{J1} &\leq \frac{1}{4}P^2z_1^2 + \frac{1}{4}\hat{\zeta}_{a1}^T\Psi_{J1}\Psi_{J1}^T\hat{\zeta}_{a1} \end{aligned} \quad (30)$$

Along with (29) and (30), we can calculate:

$$\begin{aligned} \dot{V}_1 &\leq -\chi_1z_1 + \kappa_{c1}\tilde{\zeta}_{c1}^T\Psi_{J1}\Psi_{J1}^T\hat{\zeta}_{c1} \\ &\quad + \kappa_{a1}\tilde{\zeta}_{a1}^T\Psi_{J1}\Psi_{J1}^T\hat{\zeta}_{a1} + \frac{1}{2}z_2^2 + \frac{1}{2}\dot{y}_r^2 \\ &\quad + (\kappa_{c1} - \kappa_{a1})\tilde{\zeta}_{a1}^T\Psi_{J1}\Psi_{J1}^T\hat{\zeta}_{c1} \\ &\quad + \frac{1}{4}\hat{\zeta}_{a1}^T\Psi_{J1}\Psi_{J1}^T\hat{\zeta}_{a1} + \frac{1}{2}Q^2 \end{aligned} \quad (31)$$

Based on  $\tilde{\zeta}_{c1} = \zeta_{J1}^* - \hat{\zeta}_{c1}$ ,  $\tilde{\zeta}_{a1} = \zeta_{J1}^* - \hat{\zeta}_{a1}$  and Young's inequality, we have

$$\begin{aligned} \tilde{\zeta}_{c1}^T\Psi_{J1}\Psi_{J1}^T\hat{\zeta}_{c1} &= \frac{1}{2}\zeta_{J1}^{*T}\Psi_{J1}\Psi_{J1}^T\zeta_{J1}^* - \frac{1}{2}\tilde{\zeta}_{c1}^T\Psi_{J1} \\ &\quad \times \Psi_{J1}^T\tilde{\zeta}_{c1} - \frac{1}{2}\tilde{\zeta}_{c1}^T\Psi_{J1}\Psi_{J1}^T\hat{\zeta}_{c1} \\ \tilde{\zeta}_{a1}^T\Psi_{J1}\Psi_{J1}^T\hat{\zeta}_{a1} &= \frac{1}{2}\zeta_{J1}^{*T}\Psi_{J1}\Psi_{J1}^T\zeta_{J1}^* - \frac{1}{2}\tilde{\zeta}_{a1}^T\Psi_{J1} \\ &\quad \times \Psi_{J1}^T\tilde{\zeta}_{a1} - \frac{1}{2}\tilde{\zeta}_{a1}^T\Psi_{J1}\Psi_{J1}^T\hat{\zeta}_{a1} \\ \tilde{\zeta}_{a1}^T\Psi_{J1}\Psi_{J1}^T\hat{\zeta}_{c1} &\leq -\frac{1}{2}\tilde{\zeta}_{a1}^T\Psi_{J1}\Psi_{J1}^T\tilde{\zeta}_{a1} \\ &\quad - \frac{1}{2}\tilde{\zeta}_{c1}^T\Psi_{J1}\Psi_{J1}^T\hat{\zeta}_{c1} \end{aligned} \quad (32)$$

Subsequently, we can acquire

$$\begin{aligned} \dot{V}_1 &\leq -\chi_1z_1 - \frac{\kappa_{c1}}{2}\tilde{\zeta}_{c1}^T\Psi_{J1}\Psi_{J1}^T\tilde{\zeta}_{c1} \\ &\quad - (\kappa_{a1} - \frac{\kappa_{c1}}{2})\tilde{\zeta}_{a1}^T\Psi_{J1}\Psi_{J1}^T\tilde{\zeta}_{a1} \\ &\quad - \frac{\kappa_{a1}}{2}\tilde{\zeta}_{c1}^T\Psi_{J1}\Psi_{J1}^T\hat{\zeta}_{c1} - (\frac{\kappa_{a1}}{2} - \frac{1}{4}) \\ &\quad \times \tilde{\zeta}_{a1}^T\Psi_{J1}\Psi_{J1}^T\hat{\zeta}_{a1} + \frac{1}{2}z_2^2 + \frac{1}{2}\dot{y}_r^2 \\ &\quad + \frac{\kappa_{c1} + \kappa_{a1}}{2}\zeta_{J1}^{*T}\Psi_{J1}\Psi_{J1}^T\zeta_{J1}^* + \frac{1}{2}Q^2 \end{aligned} \quad (33)$$

The following inequality holds when  $\lambda_{\Psi_{J1}}^{\min}$  is the minimum eigenvalue of  $\Psi_{J1}\Psi_{J1}^T$ .

$$\begin{aligned} -\tilde{\zeta}_{c1}^T\Psi_{J1}\Psi_{J1}^T\tilde{\zeta}_{c1} &\leq -\lambda_{\Psi_{J1}}^{\min}\tilde{\zeta}_{c1}^T\tilde{\zeta}_{c1} \\ -\tilde{\zeta}_{a1}^T\Psi_{J1}\Psi_{J1}^T\tilde{\zeta}_{a1} &\leq -\lambda_{\Psi_{J1}}^{\min}\tilde{\zeta}_{a1}^T\tilde{\zeta}_{a1} \end{aligned} \quad (34)$$

According to the design parameters  $\kappa_{a1} > \frac{\kappa_{c1}}{2}$  and  $\kappa_{a1} > \frac{1}{2}$ , as well as (34), it can yield

$$\begin{aligned} \dot{V}_1 &\leq -\chi_1z_1 - \frac{\kappa_{c1}}{2}\lambda_{\Psi_{J1}}^{\min}\tilde{\zeta}_{c1}^T\tilde{\zeta}_{c1} \\ &\quad - (\kappa_{a1} - \frac{\kappa_{c1}}{2})\lambda_{\Psi_{J1}}^{\min}\tilde{\zeta}_{a1}^T\tilde{\zeta}_{a1} + \frac{1}{2}z_2^2 + \sigma_1 \end{aligned} \quad (35)$$

where  $\sigma_1 = \frac{1}{2}\dot{y}_r^2 + \frac{1}{2}Q^2 + \frac{\kappa_{c1} + \kappa_{a1}}{2}\zeta_{J1}^{*T}\Psi_{J1}\Psi_{J1}^T\zeta_{J1}^*$ . Since all the terms in  $\sigma_1$  are bounded, there exists a positive constant  $\bar{\sigma}_1$  such that  $|\sigma_1| \leq \bar{\sigma}_1$ .

**Step 2 :** The derivative of  $z_2$  is calculated in a similar manner.

$$\dot{z}_2 = \xi_3 - \frac{B}{m}\xi_2 - F_T - mg - \dot{\alpha}_1^* \quad (36)$$

Among them,  $-\frac{B}{m}\xi_2 - F_T - mg$  can be approximated by NN as  $\zeta_{f2}^{*T}\Psi_{f2} + \varepsilon_{f2}$ , there exists a positive constant  $\bar{\varepsilon}_{f2}$  such that  $|\varepsilon_{f2}| \leq \bar{\varepsilon}_{f2}$ . Then, the selection of the most suitable integral cost function is detailed as follows:

$$\begin{aligned} J_2^*(z_2) &= \int_t^\infty h_2(z_2(v), \alpha_2^*(z_2(v)))dv \\ &= \min_{\alpha_2 \in \Omega_{z_2}} \left\{ \int_t^\infty h_2(z_2(v), \alpha_2(z_2(v)))dv \right\} \end{aligned} \quad (37)$$

where  $h_2(z_2, \alpha_2) = z_2^2 + \alpha_2^2$  is the cost function,  $\alpha_2^*$  represents the optimal controller.

Based on (37), the HJB equation is constructed as

$$\begin{aligned} H_2(z_2, \alpha_2^*, \frac{dJ_2^*}{dz_2}) &= z_2^2 + \alpha_2^{*2} + \frac{dJ_2^*}{dz_2}(\alpha_2^* + \zeta_{f2}^{*T}\Psi_{f2}(\xi) \\ &\quad + \varepsilon_f(\xi) - \dot{\alpha}_1^*) = 0 \end{aligned} \quad (38)$$

The same as before, we can solve for  $\partial H_2 / \partial \alpha_2^* = 0$  as

$$\alpha_2^* = -\frac{1}{2} \frac{dJ_2^*(z_2)}{dz_2} \quad (39)$$

Then,  $\frac{dJ_2^*(z_2)}{dz_2}$  can be factored as

$$\begin{aligned} \frac{dJ_2^*(z_2)}{dz_2} &= (2\chi_2 + \frac{9}{2})z_2 + 2\zeta_{f2}^{*T}\Psi_{f2} + 2\varepsilon_{f2} \\ &\quad + J_2^o(\xi_2, z_2) \end{aligned} \quad (40)$$

where  $\chi_2 > 0$  is design parameter.  $J_2^o(\xi_2, z_2) = -(2\chi_2 + \frac{9}{2})z_2 - 2\zeta_{f2}^{*T}\Psi_{f2} - 2\varepsilon_{f2} + \frac{dJ_2^*(z_2)}{dz_2}$  is a continuous function, and the  $\alpha_2^*$  can be expressed as

$$\alpha_2^* = -(\chi_2 + \frac{9}{4})z_2 - \zeta_{f2}^{*T}\Psi_{f2} - \varepsilon_{f2} - \frac{1}{2}J_2^o(\xi_2, z_2) \quad (41)$$

Since  $J_2^o(\xi_2, z_2)$  is unknown continuous term, it can also be approximated using NN as follows:

$$J_2^o(\xi_2, z_2) = \zeta_{J2}^{*T}\Psi_{J2} + \varepsilon_{J2} \quad (42)$$

where  $\zeta_{J2}^*$  is the ideal weight vector,  $\Psi_{J2}$  is the NN basis function vector, and the NN approximation error  $\varepsilon_{J2}$  is bounded.

Similarly, we can derive the following conclusion

$$\frac{dJ_2^*(z_2)}{dz_2} = (2\chi_2 + \frac{9}{2})z_2 + 2\zeta_{f2}^{*T}\Psi_{f2} + \zeta_{J2}^{*T}\Psi_{J2} + \varepsilon_2 \quad (43)$$

$$\alpha_2^* = -(\chi_2 + \frac{9}{4})z_2 - \zeta_{f2}^{*T}\Psi_{f2} - \frac{1}{2}\zeta_{J2}^{*T}\Psi_{J2} - \frac{1}{2}\varepsilon_2 \quad (44)$$

where  $\varepsilon_2 = 2\varepsilon_{f2} + \varepsilon_{J2}$ .

The optimal control (44), however, remains unattainable, necessitating the execution of an RL algorithm featuring both a critic and an actor to acquire viable control signal.

$$\frac{d\hat{J}_2^*(z_2)}{dz_2} = (2\chi_2 + \frac{9}{2})z_2 + 2\hat{\zeta}_{f2}^T\Psi_{f2} + \hat{\zeta}_{c2}^T\Psi_{J2} \quad (45)$$

$$\hat{\alpha}_2^* = -(\chi_2 + \frac{9}{4})z_2 - \hat{\zeta}_{f2}^T\Psi_{f2} - \frac{1}{2}\hat{\zeta}_{a2}^T\Psi_{J2} \quad (46)$$

where  $\frac{d\hat{J}_2^*(z_2)}{dz_2}$  and  $\hat{\alpha}_2^*$  are the estimate of  $\frac{dJ_2^*(z_2)}{dz_2}$  and  $\alpha_2^*$ , respectively.  $\hat{\zeta}_{c2}^T\Psi_{J2}$  and  $\hat{\zeta}_{a2}^T\Psi_{J2}$  are the NN weight vectors of critic and actor, respectively.

Same as the first step, the corresponding three adaptive update laws are designed as follows:

$$\dot{\hat{\zeta}}_{f2} = \Gamma_{f2}\Psi_{f2}z_2 - \kappa_{f2}\hat{\zeta}_{f2} \quad (47)$$

$$\dot{\hat{\zeta}}_{c2} = -\kappa_{c2}\Psi_{J2}\Psi_{J2}^T\hat{\zeta}_{c2} \quad (48)$$

$$\dot{\hat{\zeta}}_{a2} = -\Psi_{J2}\Psi_{J2}^T(\kappa_{a2}(\hat{\zeta}_{a2} - \hat{\zeta}_{c2}) + \kappa_{c2}\hat{\zeta}_{c2}) \quad (49)$$

where  $\Gamma_{f2} > 0$ ,  $\kappa_{f2} > 0$ ,  $\kappa_{c2} > 0$  and  $\kappa_{a2} > 0$  are design parameters, while  $\kappa_{c2}$  and  $\kappa_{a2}$  satisfy  $\kappa_{a2} > \frac{1}{2}$ ,  $\kappa_{a2} > \frac{\kappa_{c2}}{2}$ .

According to (46), the  $\dot{z}_2$  can be expressed as follows

$$\begin{aligned} \dot{z}_2 = & -(\chi_2 + \frac{9}{4})z_2 + z_3 - \frac{1}{2}\hat{\zeta}_{a2}^T\Psi_{J2} \\ & + \hat{\zeta}_{f2}^T\Psi_{f2} + \varepsilon_{f2} - \dot{\hat{\alpha}}_1^* \end{aligned} \quad (50)$$

Subsequently, the Lyapunov function  $V_2$  is established as

$$V_2 = \frac{1}{2}z_2^2 + \frac{1}{2\Gamma_{f2}}\tilde{\zeta}_{f2}^T\tilde{\zeta}_{f2} + \frac{1}{2}\tilde{\zeta}_{c2}^T\tilde{\zeta}_{c2} + \frac{1}{2}\tilde{\zeta}_{a2}^T\tilde{\zeta}_{a2} \quad (51)$$

where  $\tilde{\zeta}_{f2} = \zeta_{f2}^* - \hat{\zeta}_{f2}$ ,  $\tilde{\zeta}_{c2} = \zeta_{c2}^* - \hat{\zeta}_{c2}$  and  $\tilde{\zeta}_{a2} = \zeta_{a2}^* - \hat{\zeta}_{a2}$ .

Then, the  $\dot{V}_2$  can be calculated as

$$\begin{aligned} \dot{V}_2 = & z_2(-(\chi_2 + \frac{9}{4})z_2 + z_3 - \frac{1}{2}\hat{\zeta}_{a2}^T\Psi_{J2} + \hat{\zeta}_{f2}^T\Psi_{f2} \\ & + \varepsilon_{f2} - \dot{\hat{\alpha}}_1^*) + \frac{\kappa_{f2}}{\Gamma_{f2}}\tilde{\zeta}_{f2}^T\hat{\zeta}_{f2} + \kappa_{c2}\tilde{\zeta}_{c2}^T\Psi_{J2}\Psi_{J2}^T\hat{\zeta}_{c2} \\ & + \tilde{\zeta}_{a2}^T\Psi_{J2}\Psi_{J2}^T(\kappa_{a2}(\hat{\zeta}_{a2} - \hat{\zeta}_{c2}) + \kappa_{c2}\hat{\zeta}_{c2}) \end{aligned} \quad (52)$$

Using the Young's inequality, we have

$$\begin{aligned} z_2z_3 & \leq \frac{1}{2}z_2^2 + \frac{1}{2}z_3^2 \\ z_2\varepsilon_{f2} & \leq \frac{1}{2}z_2^2 + \frac{1}{2}\varepsilon_{f2}^2 \\ -z_2\dot{\hat{\alpha}}_1^* & \leq \frac{1}{2}z_2^2 + \frac{1}{2}\dot{\hat{\alpha}}_1^{*2} \\ -\frac{1}{2}z_2\hat{\zeta}_{a2}^T\Psi_{J2} & \leq \frac{1}{4}z_2^2 + \frac{1}{4}\hat{\zeta}_{a2}^T\Psi_{J2}\Psi_{J2}^T\hat{\zeta}_{a2} \end{aligned} \quad (53)$$

Substituting (53) into (52) yields

$$\begin{aligned} \dot{V}_2 \leq & -\chi_2z_2^2 - \frac{\kappa_{f2}}{2\Gamma_{f2}}\tilde{\zeta}_{f2}^T\tilde{\zeta}_{f2} - \frac{\kappa_{c2}}{2}\tilde{\zeta}_{c2}^T\Psi_{J2}\Psi_{J2}^T\tilde{\zeta}_{c2} \\ & -(\kappa_{a2} - \frac{\kappa_{c2}}{2})\tilde{\zeta}_{a2}^T\Psi_{J2}\Psi_{J2}^T\tilde{\zeta}_{a2} - \frac{\kappa_{a2}}{2}\hat{\zeta}_{c2}^T\Psi_{J2}\Psi_{J2}^T\hat{\zeta}_{c2} \\ & -(\frac{\kappa_{a2}}{2} - \frac{1}{4})\hat{\zeta}_{a2}^T\Psi_{J2}\Psi_{J2}^T\hat{\zeta}_{a2} + \frac{\kappa_{c2} + \kappa_{a2}}{2}\hat{\zeta}_{c2}^T\Psi_{J2} \\ & \times \Psi_{J2}^T\zeta_{a2}^* + \frac{1}{2}\varepsilon_{f2}^2 + \frac{1}{2}\dot{\hat{\alpha}}_1^{*2} + \frac{\kappa_{f2}}{2}\zeta_{f2}^{*T}\zeta_{f2}^* + \frac{1}{2}z_3^2 \\ \leq & -\chi_2z_2^2 - \frac{\kappa_{f2}}{2\Gamma_{f2}}\tilde{\zeta}_{f2}^T\tilde{\zeta}_{f2} - \frac{\kappa_{c2}}{2}\lambda_{\Psi_{J2}}^{\min}\tilde{\zeta}_{c2}^T\tilde{\zeta}_{c2} \\ & -(\kappa_{a2} - \frac{\kappa_{c2}}{2})\lambda_{\Psi_{J2}}^{\min}\tilde{\zeta}_{a2}^T\tilde{\zeta}_{a2} - \frac{1}{2}z_2^2 + \frac{1}{2}z_3^2 + \sigma_2 \end{aligned} \quad (54)$$

where  $\sigma_2 = \frac{1}{2}\varepsilon_{f2}^2 + \frac{\kappa_{c2} + \kappa_{a2}}{2}\zeta_{a2}^{*T}\Psi_{J2}\Psi_{J2}^T\zeta_{a2}^* + \frac{\kappa_{f2}}{2}\zeta_{f2}^{*T}\zeta_{f2}^* + \frac{1}{2}\dot{\hat{\alpha}}_1^{*2}$  is bounded, and there exists a positive constant  $\bar{\sigma}_2$  that ensures the existence of  $|\sigma_2| \leq \bar{\sigma}_2$ . Additionally,  $\lambda_{\Psi_{J2}}^{\min}$  represents the minimum eigenvalue of  $\Psi_{J2}\Psi_{J2}^T$ .

**Step 3 :** Similarly, the derivative of  $z_3$  is

$$\begin{aligned} \dot{z}_3 = & \dot{\xi}_3 - \dot{\hat{\alpha}}_2^* \\ = & \gamma_1u - \gamma_2\xi_2 - \gamma_3(p_1 - p_2) - \dot{\hat{\alpha}}_2^* \end{aligned} \quad (55)$$

where  $-\gamma_2\xi_2 - \gamma_3(p_1 - p_2)$  can be approximated by NN as  $\hat{\zeta}_{f3}^T\Psi_{f3} + \varepsilon_{f3}$ , there exists a positive constant  $\bar{\varepsilon}_{f3}$  such that  $|\varepsilon_{f3}(\xi)| \leq \bar{\varepsilon}_{f3}$ . Then, the selection of the most suitable integral cost function is detailed as follows:

$$\begin{aligned} J_3^*(z_3) = & \int_t^\infty h_3(z_3(v), u^*(z_3(v)))dv \\ = & \min_{u \in \Omega_{z_3}} \left\{ \int_t^\infty h_3(z_3(v), u(z_3(v)))dv \right\} \end{aligned} \quad (56)$$

where  $h_3(z_3, u) = z_3^2 + u^2$  is the cost function,  $u^*$  represents the optimal controller.

Based on (56), the HJB equation is constructed as

$$\begin{aligned} H_3(z_3, u^*, \frac{dJ_3^*}{dz_3}) = & z_3^2 + u^{*2} + \frac{dJ_3^*}{dz_3}(u^* + \hat{\zeta}_{f3}^T\Psi_{f3} \\ & + \varepsilon_{f3} - \dot{\hat{\alpha}}_2^*) = 0 \end{aligned} \quad (57)$$

The same as before, we can solve for  $\partial H_3/\partial u^* = 0$  as

$$u^* = -\frac{1}{2}\frac{dJ_3^*(z_3)}{dz_3} \quad (58)$$

Then,  $\frac{dJ_3^*(z_3)}{dz_3}$  can be factored as

$$\frac{dJ_3^*(z_3)}{dz_3} = (2\chi_3 + \frac{7}{2})z_3 + 2\hat{\zeta}_{f3}^{*T}\Psi_{f3} + 2\varepsilon_{f3} + J_3^o(\xi_3, z_3) \quad (59)$$

where  $\chi_3 > 0$  is design parameter.  $J_3^o(\xi_3, z_3) = -(2\chi_3 + \frac{7}{2})z_3 - 2\hat{\zeta}_{f3}^{*T}\Psi_{f3} - 2\varepsilon_{f3} + \frac{dJ_3^*(z_3)}{dz_3}$  is a continuous function, and the  $u^*$  can be expressed as

$$u^* = -(\chi_3 + \frac{7}{4})z_3 - \hat{\zeta}_{f3}^{*T}\Psi_{f3} - \varepsilon_{f3} - \frac{1}{2}J_3^o(\xi_3, z_3) \quad (60)$$

Since  $J_3^o(\xi_3, z_3)$  is unknown continuous term, it can also be approximated using NN as follows:

$$J_3^o(\xi_3, z_3) = \hat{\zeta}_{J3}^{*T}\Psi_{J3} + \varepsilon_{J3} \quad (61)$$

where  $\hat{\zeta}_{J3}^*$  is the ideal weight vector,  $\Psi_{J3}$  is the NN basis function vector, and the NN approximation error  $\varepsilon_{J3}$  is bounded.

Similarly, we can derive the following conclusion

$$\frac{dJ_3^*(z_3)}{dz_3} = (2\chi_3 + \frac{7}{2})z_3 + 2\hat{\zeta}_{f3}^{*T}\Psi_{f3} + \hat{\zeta}_{J3}^{*T}\Psi_{J3} + \varepsilon_3 \quad (62)$$

$$u^* = -(\chi_3 + \frac{7}{4})z_3 - \hat{\zeta}_{f3}^{*T}\Psi_{f3} - \frac{1}{2}\hat{\zeta}_{J3}^{*T}\Psi_{J3} - \frac{1}{2}\varepsilon_3 \quad (63)$$

where  $\varepsilon_3 = 2\varepsilon_{f3} + \varepsilon_{J3}$ . For (63), however, remains unattainable, necessitating the execution of an RL algorithm featuring both a critic and an actor to acquire viable control signal.

$$\frac{d\hat{J}_3^*(z_3)}{dz_3} = (2\chi_3 + \frac{7}{2})z_3 + 2\hat{\zeta}_{f3}^T\Psi_{f3} + \hat{\zeta}_{c3}^T\Psi_{J3} \quad (64)$$

$$\hat{u}^* = -(\chi_3 + \frac{7}{4})z_3 - \hat{\zeta}_{f3}^T\Psi_{f3} - \frac{1}{2}\hat{\zeta}_{a3}^T\Psi_{J3} \quad (65)$$

where  $\frac{d\hat{J}_3^*(z_3)}{dz_3}$  and  $\hat{u}^*$  are the estimate of  $\frac{dJ_3^*(z_3)}{dz_3}$  and  $u^*$ , respectively.  $\hat{\zeta}_{c3}^T\Psi_{J3}$  and  $\hat{\zeta}_{a3}^T\Psi_{J3}$  are the NN weight vectors of critic and actor, respectively.

Then, the corresponding three adaptive update laws are designed as follows:

$$\dot{\hat{\zeta}}_{f3} = \Gamma_{f3}\Psi_{f3}z_3 - \kappa_{f3}\hat{\zeta}_{f3} \quad (66)$$

$$\dot{\hat{\zeta}}_{c3} = -\kappa_{c3}\Psi_{J3}\Psi_{J3}^T\hat{\zeta}_{c3} \quad (67)$$

$$\dot{\hat{\zeta}}_{a3} = -\Psi_{J3}\Psi_{J3}^T(\kappa_{a3}(\hat{\zeta}_{a3} - \hat{\zeta}_{c3}) + \kappa_{c3}\hat{\zeta}_{c3}) \quad (68)$$

where  $\Gamma_{f3} > 0$ ,  $\kappa_{f3} > 0$ ,  $\kappa_{c3} > 0$  and  $\kappa_{a3} > 0$  are design parameters, while  $\kappa_{c3}$  and  $\kappa_{a3}$  satisfy  $\kappa_{a3} > \frac{\gamma_1^2}{2}$ ,  $\kappa_{a3} > \frac{\kappa_{c3}}{3}$ .

Following (55) and (65), we obtain  $\dot{z}_3$

$$\dot{z}_3 = \gamma_1(-(\chi_3 + \frac{7}{4})z_3 - \frac{1}{2}\hat{\zeta}_{a3}^T\Psi_{J3} + \hat{\zeta}_{f3}^T\Psi_{f3} + \varepsilon_{f3} - \dot{\hat{\alpha}}_1^*) \quad (69)$$

Subsequently, the Lyapunov function  $V_3$  is established as

$$V_3 = \frac{1}{2}z_3^2 + \frac{1}{2\Gamma_{f3}}\tilde{\zeta}_{f3}^T\tilde{\zeta}_{f3} + \frac{1}{2}\tilde{\zeta}_{c3}^T\tilde{\zeta}_{c3} + \frac{1}{2}\tilde{\zeta}_{a3}^T\tilde{\zeta}_{a3} \quad (70)$$

where  $\tilde{\zeta}_{f3} = \zeta_{f3}^* - \hat{\zeta}_{f3}$ ,  $\tilde{\zeta}_{c3} = \zeta_{c3}^* - \hat{\zeta}_{c3}$  and  $\tilde{\zeta}_{a3} = \zeta_{a3}^* - \hat{\zeta}_{a3}$ . Then, the  $\dot{V}_3$  can be calculated as

$$\begin{aligned} \dot{V}_3 = & \gamma_1 z_3 \left( -(\chi_3 + \frac{7}{4})z_3 - \frac{1}{2}\hat{\zeta}_{a3}^T\Psi_{J3} + \tilde{\zeta}_{f3}^T\Psi_{f3} + \varepsilon_{f3} \right. \\ & \left. - \dot{\alpha}_2^* \right) + \frac{\kappa_{f3}}{\Gamma_{f3}}\tilde{\zeta}_{f3}^T\hat{\zeta}_{f3} + \kappa_{c3}\tilde{\zeta}_{c3}^T\Psi_{J3}\Psi_{J3}^T\hat{\zeta}_{c3} \\ & + \tilde{\zeta}_{a3}^T\Psi_{J3}\Psi_{J3}^T(\kappa_{a3}(\hat{\zeta}_{a3} - \hat{\zeta}_{c3}) + \kappa_{c3}\hat{\zeta}_{c3}) \end{aligned} \quad (71)$$

Using the Young's inequality, we have

$$\begin{aligned} \gamma_1 z_3 \varepsilon_{f3} & \leq \frac{1}{2}z_3^2 + \frac{\gamma_1^2}{2}\varepsilon_{f3}^2 \\ -\gamma_1 z_3 \dot{\alpha}_2^* & \leq \frac{1}{2}z_3^2 + \frac{\gamma_1^2}{2}\dot{\alpha}_2^{*2} \\ -\frac{1}{2}\gamma_1 z_3 \hat{\zeta}_{a3}^T\Psi_{J3} & \leq \frac{1}{4}z_3^2 + \frac{\gamma_1^2}{4}\hat{\zeta}_{a3}^T\Psi_{J3}\Psi_{J3}^T\hat{\zeta}_{a3} \end{aligned} \quad (72)$$

Substituting (72) into (71) yields

$$\begin{aligned} \dot{V}_3 \leq & -\chi_3 z_3^2 - \frac{\kappa_{f3}}{2\Gamma_{f3}}\tilde{\zeta}_{f3}^T\tilde{\zeta}_{f3} - \frac{\kappa_{c3}}{2}\tilde{\zeta}_{c3}^T\Psi_{J3}\Psi_{J3}^T\tilde{\zeta}_{c3} \\ & - (\kappa_{a3} - \frac{\kappa_{c3}}{2})\tilde{\zeta}_{a3}^T\Psi_{J3}\Psi_{J3}^T\tilde{\zeta}_{a3} - \frac{\kappa_{a3}}{2}\hat{\zeta}_{c3}^T\Psi_{J3}\Psi_{J3}^T\hat{\zeta}_{c3} \\ & - (\frac{\kappa_{a3}}{2} - \frac{\gamma_1^2}{4})\hat{\zeta}_{a3}^T\Psi_{J3}\Psi_{J3}^T\hat{\zeta}_{a3} + \frac{\kappa_{c3} + \kappa_{a3}}{2}\zeta_{J3}^{*T}\Psi_{J3} \\ & \times \Psi_{J3}^T\zeta_{J3}^* + \frac{1}{2}\varepsilon_{f3}^2 + \frac{1}{2}\dot{\alpha}_2^{*2} + \frac{\kappa_{f3}}{2}\zeta_{f3}^{*T}\zeta_{f3}^* - \frac{1}{2}z_3^2 + \sigma_3 \\ \leq & -\chi_3 z_3^2 - \frac{\kappa_{f3}}{2\Gamma_{f3}}\tilde{\zeta}_{f3}^T\tilde{\zeta}_{f3} - \frac{\kappa_{c3}}{2}\lambda_{\Psi_{J3}}^{\min}\tilde{\zeta}_{c3}^T\tilde{\zeta}_{c3} \\ & - (\kappa_{a3} - \frac{\kappa_{c3}}{2})\lambda_{\Psi_{J3}}^{\min}\tilde{\zeta}_{a3}^T\tilde{\zeta}_{a3} - \frac{1}{2}z_3^2 + \sigma_3 \end{aligned} \quad (73)$$

where  $\sigma_3 = \frac{\gamma_1^2}{2}\varepsilon_{f3}^2 + \frac{\kappa_{c3} + \kappa_{a3}}{2}\zeta_{J3}^{*T}\Psi_{J3}\Psi_{J3}^T\zeta_{J3}^* + \frac{\gamma_1^2}{2}\dot{\alpha}_2^{*2} + \frac{\kappa_{f3}}{2}\zeta_{f3}^{*T}\zeta_{f3}^*$  is bounded, and there exists a positive constant  $\bar{\sigma}_3$  that ensures the existence of  $|\sigma_3| \leq \bar{\sigma}_3$ . Additionally,  $\lambda_{\Psi_{J3}}^{\min}$  represents the minimum eigenvalue of  $\Psi_{J3}\Psi_{J3}^T$ .

### B. Stability analysis

**Theorem 1** Apply the control strategy proposed in this paper to (2), where the adaptive laws for NN parameters, critics and actors are (47), (66) and (25), (48), (67) and (26), (49), (68), respectively, and the optimal virtual control and control input are (24), (46), and (65). Based on this, the following conclusions can be drawn: 1) This control strategy can ensure that all signals in the closed-loop system remain bounded and achieve the performance optimization of each subsystem; 2) The output error strictly follows the design form defined in (3) and no violation of constraints occurs.

*Proof:* Construct a Lyapunov function  $V = \sum_{i=1}^3 V_i$ , and by integrating the preceding steps, we can compute

$$\begin{aligned} \dot{V} \leq & -\sum_{i=1}^3 \chi_i z_i^2 - \sum_{i=2}^3 \frac{\kappa_{fi}}{2\Gamma_{fi}}\tilde{\zeta}_{fi}^T\tilde{\zeta}_{fi} - \sum_{i=1}^3 \frac{\kappa_{ci}}{2}\lambda_{\Psi_{Ji}}^{\min}\tilde{\zeta}_{ci}^T\tilde{\zeta}_{ci} \\ & - \sum_{i=1}^3 (\kappa_{ai} - \frac{\kappa_{ci}}{2})\lambda_{\Psi_{Ji}}^{\min}\tilde{\zeta}_{ai}^T\tilde{\zeta}_{ai} + \sum_{i=1}^3 \bar{\sigma}_i \\ \leq & -\Theta V + \Delta \end{aligned} \quad (74)$$

where  $\Theta = \min\{2\chi_i, \frac{\kappa_{fi}}{\Gamma_{fi}}, \kappa_{ci}, (\kappa_{ai} - \frac{\kappa_{ci}}{2})\lambda_{\Psi_{Ji}}^{\min}, i=1, 2, 3, j=2, 3\}$ ,  $\Delta = \sum_{i=0}^3 \bar{\sigma}_i$ . ■

The proof of Theorem 1 is completed.

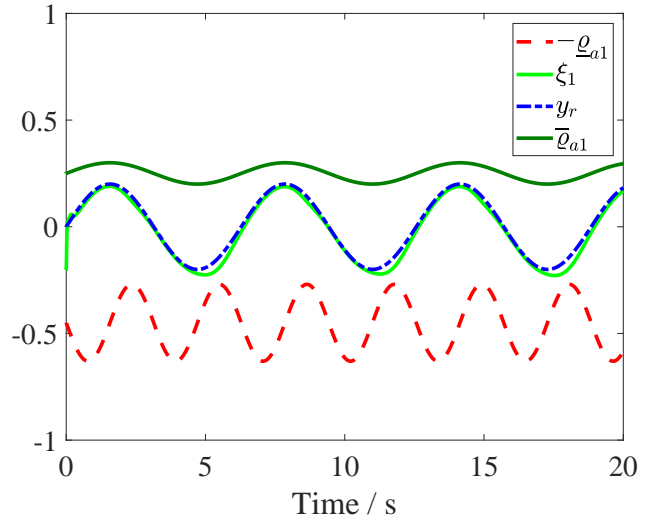


Fig. 2:  $\xi_1$ ,  $y_r$  and the constraints  $-\underline{\rho}_{a1}$ ,  $\bar{\rho}_{a1}$ .

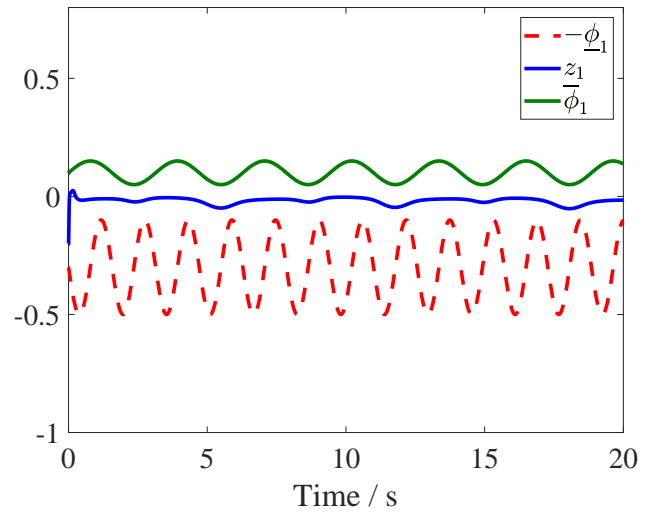
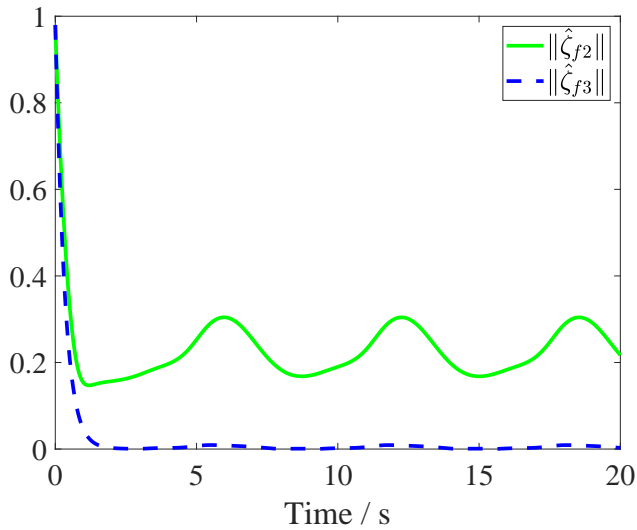
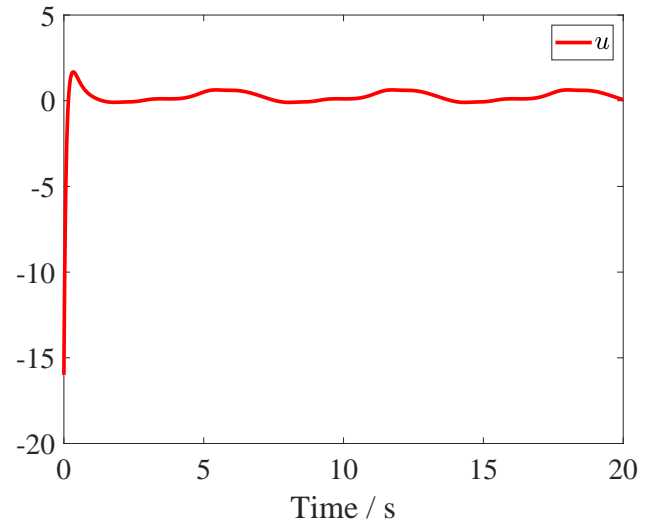
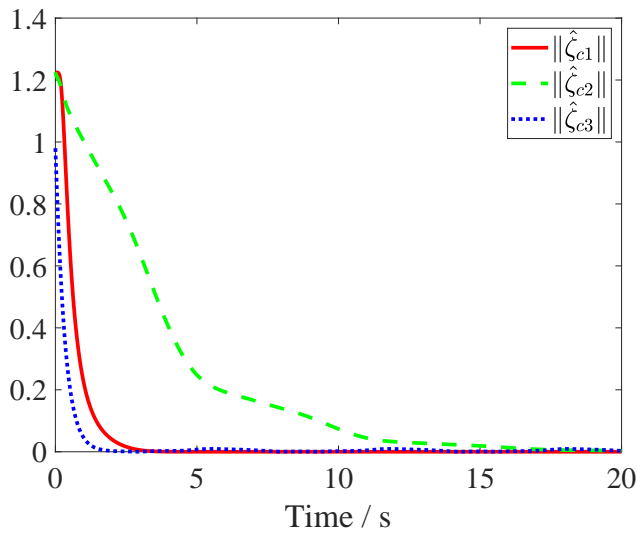
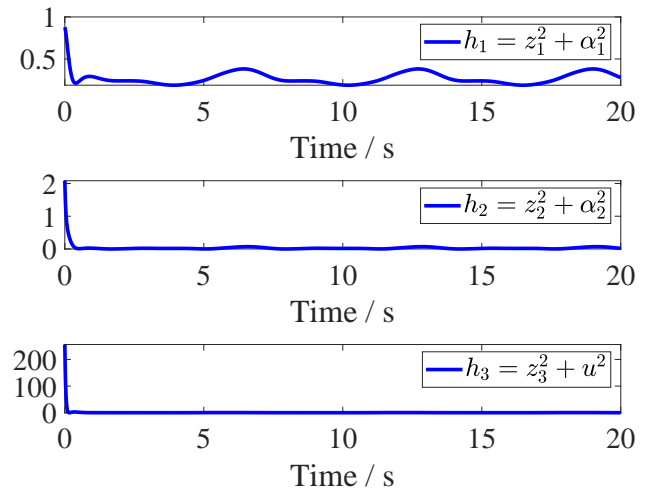
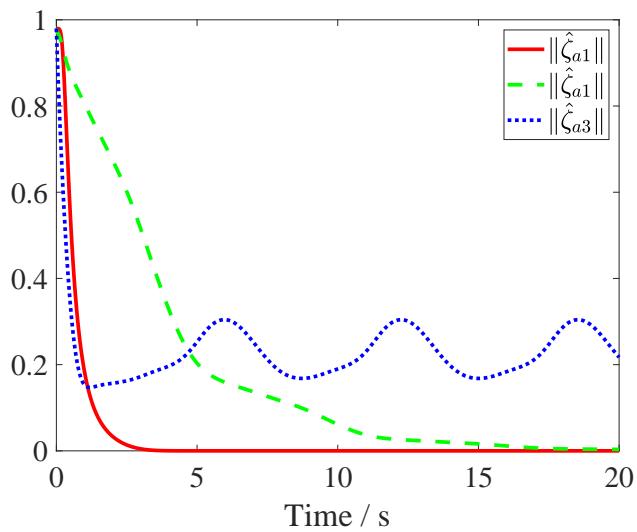


Fig. 3:  $z_1$  and the constraints  $\underline{\phi}_1$ ,  $\bar{\phi}_1$ .

## IV. SIMULATION EXAMPLE

To verify the effectiveness of the control algorithm proposed in this paper, numerical simulation verification was carried out with the aid of MATLAB. The parameters used in the simulation process are summarized as follows:

The corresponding process parameters in the electro-hydraulic control cylinder system of the hydraulic support are  $m = 300 \text{ kg}$ ,  $B = 1000 \text{ N}/(\text{m} \cdot \text{s}^{-1})$ ,  $A_1 = 1.92625 \times 10^{-3} \text{ m}^2$ ,  $A_2 = 9.4514 \times 10^{-4} \text{ m}^2$ ,  $p_s = 2 \times 10^7 \text{ Pa}$ ,  $p_r = 0$ ,  $k_q k_v = 8.9 \times 10^{-8} \text{ m}^3/(\text{s} \cdot \text{V} \cdot \sqrt{\text{Pa}})$ ,  $\beta_e = 7 \times 10^8 \text{ Pa}$ ,  $C_t = 4 \times 10^{-13} \text{ m}^3/(\text{s} \cdot \text{Pa})$ . The output  $\xi_1$  is constrained within the range of  $-0.18 \sin(2t) - 0.45 < \xi_1 < 0.05 \sin(t) + 0.25$ . The reference signal is selected as  $y_r = 0.2 \sin(t)$ .


 Fig. 4: The norms of the  $\hat{\zeta}_{f2}$  and  $\hat{\zeta}_{f3}$ .

 Fig. 7: Control input  $u$ .

 Fig. 5: The norms of the  $\hat{\zeta}_{c1}$ ,  $\hat{\zeta}_{c2}$  and  $\hat{\zeta}_{c3}$ .

 Fig. 8: Cost functions  $h_1$ ,  $h_2$  and  $h_3$ .

 Fig. 6: The norms of the  $\hat{\zeta}_{a1}$ ,  $\hat{\zeta}_{a2}$  and  $\hat{\zeta}_{a3}$ .

The control parameters are designed as  $\chi_1 = 8$ ,  $\chi_2 = 12$ ,  $\chi_3 = 18$ ,  $\kappa_{f2} = 15$ ,  $\kappa_{f3} = 20$ ,  $\kappa_{c1} = \kappa_{c2} = \kappa_{c3} = 10$ ,  $\kappa_{a1} = \kappa_{a2} = \kappa_{a3} = 12$ . Furthermore, the initial values are set as  $\xi_1(0) = -0.2$ ,  $\xi_2(0) = \xi_3(0) = 0.2$ ,  $\hat{\zeta}_{f2}(0) = \hat{\zeta}_{f3}(0) = [0.2, \dots, 0.2]^T \in \mathbb{R}^{6 \times 1}$ ,  $\hat{\zeta}_{c1}(0) = \hat{\zeta}_{a1}(0) = [0.5, \dots, 0.5]^T \in \mathbb{R}^{6 \times 1}$ ,  $\hat{\zeta}_{c2}(0) = \hat{\zeta}_{a2}(0) = [0.4, \dots, 0.4]^T \in \mathbb{R}^{6 \times 1}$ ,  $\hat{\zeta}_{c3}(0) = \hat{\zeta}_{a3}(0) = [0.4, \dots, 0.4]^T \in \mathbb{R}^{6 \times 1}$ .

The simulation results show that the dual neural network structure based on the critic-actor framework proposed in this paper can efficiently evaluate the value function of the current control strategy, generate adaptive compensation terms to optimize the control law, and ensure that the output error of the hydraulic support cylinder system is always within the preset range by dynamically adjusting the system control gain online. Figures 2 and 3 verify the excellent tracking performance of the system under this strategy. Figures 4 to 8 further demonstrate that the critic adaptive law, the actor adaptive law, and the optimal controller designed in this paper all exhibit fast convergence characteristics and maintain stability, thereby effectively achieving the optimal control state of the system.

## V. CONCLUSION

Based on the working principle of the hydraulic support electro-hydraulic control system, a more precise control mathematical model was constructed. Combined with reinforcement learning technology, an optimal backstepping controller with a critic-actor mechanism was designed. By introducing the asymmetric constraint theory method, this controller not only achieved performance optimization of each subsystem but also effectively ensured that the output error always met the preset asymmetric constraint conditions. Simulation results show that the proposed method significantly improves the robustness and convergence efficiency of the system, fully verifying the effectiveness and engineering feasibility of the control strategy. This research provides a solid theoretical support and feasible practical solution for the performance optimization of the HSCS, and has the potential for further application verification in complex industrial scenarios.

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